

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE
BEFORE THE BOARD OF APPEALS AND INTERFERENCES

In re Patent Application of	Confirmation No.: 5248
Jan WEBJÖRN	Date: March 18, 2010
Serial No.: 10/500,583	Group Art Unit: 3679
Filed: June 29, 2004	Examiner: Nahid Amiri
For: A FLANGED MEMBER AND A JOINT COMPRISING FLANGED MEMBERS	

VIA EFS-WEB
Mail Stop Appeal Brief - Patents
Commissioner for Patents
P.O. Box 1450
Alexandria, VA 22313-1450

RESPONSE TO THE NOTIFICATION OF NON-COMPLIANT APPEAL BRIEF

Sir:

In support of the Notice of Appeal filed December 3, 2009 and the Appeal Brief filed February 2, 2010, this Response is submitted in response to the Notification of Non-Compliant Appeal Brief mailed February 22, 2010. This Response is believed to comply with all of the requirements of 37 C.F.R. §41.37(c).

V. SUMMARY OF CLAIMED SUBJECT MATTER:

By way of introduction, in a pipe system or other pressure equipment device parts are sometimes joined using ring-shaped collars or flanges, as explained, for example at Specification, page 1, lines 8-22. Pressure, stresses on the parts over time may cause leakage or failure of parts. Sometimes, deformation of a part due to pressure can cause bulging of a flanged part causing it to become convex to some degree as explained, for example at Appellant's Specification, page 3, lines 11-26).

According to an aspect of Appellant's invention as claimed in claim 1 described below, the flanged member is manufactured to have a concave and inclined load transferring surface of the flanged end so that if a deformation does occur due to pressure or stress bulging out would be compensated for or suppressed, as discussed, for example, at Specification, page 4, lines 17-29.

Claim 1 is directed to a flanged member 1 that is configured to be included as a first flanged member 1 in a flanged joint in a pressure equipment device (Specification, page 1, lines 23-26; Drawings, Fig. 1). Claim 1 recites that the first flanged member 1 includes a first flanged end 5 with a first end surface 10 configured to face the second end surface 11 of the second flanged end 6 of the second flanged member 2 (Specification, page 7, lines 8-13; Drawings, Fig. 1).

Further, the first end surface includes a load transferring surface (referred to in claim 1 as a first load transferring surface) that in an unstressed condition is concave in a radial direction, is curved and defined by a concave curve function (Specification, page 7, line 28 - page 8, line 15; Drawings, Fig. 2). A proximal and a distal point on the load transferring surface, points a and b respectively, meet a plane inclined in the radial direction (Specification, page 7, line 29 - page 8, line 12; Drawings, Fig. 2 plane x).

Claim 1 also recites an innermost abutment point and an outermost abutment point of the load transferring surface, the outermost abutment point being the point situated farthest in the radial direction from the central axis of the first flanged member and the innermost abutment point being the abutment point situated nearest in the radial direction from the central axis of the first flanged member, and a boring 13 that passes between the innermost abutment point and the outermost abutment point (Specification, page 8, lines 5-12; Drawings, Fig. 2).

According to an aspect of Appellant's invention as claimed in claim 11 described below, the flanged member is manufactured to have a concave and inclined load transferring surface of

the flanged end so that if a deformation does occur due to pressure or stress bulging out would be compensated for or suppressed, as discussed, for example, at Specification, page 4, lines 17-29.

Claim 11 is directed to a joint comprising a first flanged member 1 and a second flanged member 2 adapted for a pressure equipment device (Specification, page 1, lines 23-26; Drawings, Fig. 1). Claim 11 recites that the first flanged member 1 includes a first flanged end 5 with a first end surface 10 configured to face the second end surface 11 of the second flanged end 6 of the second flanged member 2 (Specification, page 7, lines 8-13; Drawings, Fig. 1).

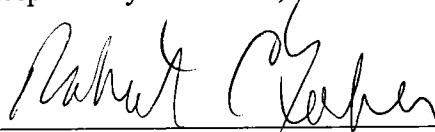
The first end surface includes a load transferring surface (referred to in claim 1 as a first load transferring surface) that in an unstressed condition is concave in a radial direction, is curved and defined by a concave curve function (Specification, page 7, line 28 - page 8, line 15; Drawings, Fig. 2). A proximal and a distal point on the load transferring surface, points a and b respectively, meet a plane inclined in the radial direction (Specification, page 7, line 29 - page 8, line 12; Drawings, Fig. 2 plane x).

Further, claim 11 recites an innermost abutment point and an outermost abutment point of the load transferring surface, the outermost abutment point being the point situated farthest in the radial direction from the central axis of the first flanged member and the innermost abutment point being the abutment point situated nearest in the radial direction from the central axis of the first flanged member, and a boring 13 that passes between the innermost abutment point and the outermost abutment point (Specification, page 8, lines 5-12; Drawings, Fig. 2).

In the event the actual fee is greater than the payment submitted or is inadvertently not enclosed or if any additional fee during the prosecution of this application is not paid, the Patent Office is authorized to charge the underpayment to Deposit Account No. 15-0700.

THIS CORRESPONDENCE IS BEING
SUBMITTED ELECTRONICALLY
THROUGH THE UNITED STATES
PATENT AND TRADEMARK OFFICE
EFS FILING SYSTEM
ON MARCH 18, 2010

Respectfully submitted,



Robert C. Faber
Registration No.: 24,322
OSTROLENK FABER LLP
1180 Avenue of the Americas
New York, New York 10036-8403
Telephone: (212) 382-0700

RCF:GB/jl